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71 Applicant: **GENERAL MOTORS CORPORATION**
 General Motors Building 3044 West Grand
 Boulevard

Detroit Michigan 48202(US)

72 Inventor: **Decker, David Michael**
 3540 St Thomas Boulevard
 Indianapolis Indiana 46214(US)

74 Representative: **Haines, Arthur Donald et al**
 Patent Section Vauxhall Motors Limited 1st
 Floor Gideon House 26 Chapel Street
 Luton, Bedfordshire LU1 2SE(GB)

54 Oil-metering system.

57 A cylindrical oil distributor insert (58) moulded of flexible and resilient fluorocarbon elastomer is self-retaining and self-sealing on an inside cylindrical wall (36) at a hollow end (16) of a rotatable shaft (18). The cylindrical wall (60) of the insert (58) has large and small perforations (72b, 74b, 74c) formed therein which define oil-catching pockets exposed to an oil spray from a stationary nozzle (52) spraying

oil into the hollow end (16) of the shaft (18). The shaft (18) has drains (86) to each oil-catching pocket large enough to exhaust oil from the pockets at rates exceeding the rates at which the pockets capture oil from the spray so that the rates at which the oil is exhausted from the oil-catching pockets are proportional to the areas of the pockets exposed to the spray.

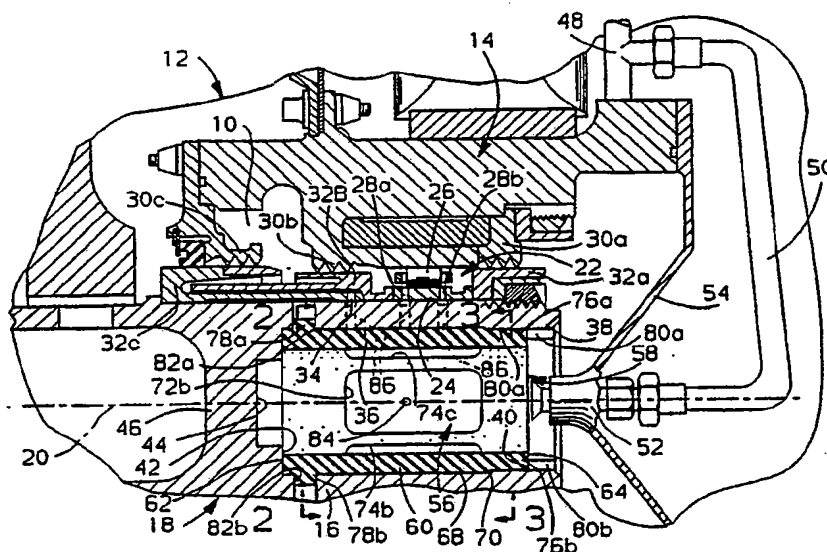


FIG. 1

Field of the Invention

This invention relates to systems for metering oil flow to rotating elements in gas turbine engines as specified in the preamble of claim 1, for example as disclosed in US-A-2,911,267.

Background of the Invention

A recognized method for cooling and lubricating gas turbine engine bearings and seal runners includes the steps of spraying oil onto an inside surface of a rotating shaft and conducting the oil to the bearings and seal runners through orifices in the shaft. Forces on the oil resulting from shaft rotation adhere the oil to the shaft as a thin film on the inside surface and propel the oil through the orifices to the seal runners and the bearings. In practicing this method, several systems have been proposed for metering or distributing the oil between the bearings and seal runners. In one system, the orifices meter the flow. In another system, different-sized pockets are formed on the inside surface of the rotating shaft to capture oil in proportion to the areas thereof which are exposed to the oil spray. In the first mentioned system, the orifices may be so small as to be susceptible to clogging by debris in the oil. In the second-mentioned system, difficult machining operations may be required if the pockets are formed directly on the shaft or a weight penalty may be experienced if the pockets are formed on a metal insert installed on the inside surface of the shaft. In the latter arrangement, seal rings between the shaft and the insert are additional parts required to practice the method.

Summary of the Invention

A metering system in a lubrication system for a gas turbine engine according to the present invention is characterised by the features specified in the characterising portion of claim 1.

This invention is a new and improved oil-metering system for conducting oil from a spray inside a rotating shaft to rotating elements on the shaft. The metering system according to this invention includes a cylindrical insert moulded of resilient and flexible fluorocarbon elastomer and including a plurality of different-sized perforations in the wall of the cylinder. The outside diameter of the insert is greater than the inside diameter of the shaft so that the insert is self-retaining. In addition, the resilience of the insert presses the outside thereof against the inside of the shaft so that the insert is self-sealing against the shaft around oil-catching pockets defined by the perforations in the cylindrical wall of

the insert. To prevent torsional distortion of the insert during shaft rotation, moulded lugs on the insert engage appropriate receptacles on the shaft.

A metering system according to this invention implements the aforesaid cooling and lubricating method with a flexible, resilient insert which co-operates with the shaft in defining the oil pockets but which is lighter than prior metal inserts and requires no retention elements and no seal rings.

Brief Description of the Drawings

Figure 1 is a fragmentary sectional view of a gas turbine engine having a rotating shaft and an oil-metering system according to this invention;

Figure 2 is a cross-sectional view taken generally along the plane indicated by lines 2-2 in Figure 1;

Figure 3 is a cross-sectional view taken generally along the plane indicated by lines 3-3 in Figure 1; and

Figure 4 is a perspective view of a portion of the oil-metering system according to this invention.

Description of a Preferred Embodiment

Referring to Figure 1, an end bearing sump 10 of a fragmentarily-illustrated gas turbine engine 12 is defined radially inboard of a rigid annular sump housing 14 and radially outboard of a hollow, tubular end 16 of a rotor shaft 18 of the engine. The tubular end of the rotor shaft is supported on the sump housing 14 for rotation about a main axis 20 of the engine by a roller bearing 22. The bearing 22 includes an inner race 24 rotatable with the shaft 18 and a plurality of rollers, only a single roller 26 being illustrated in Figure 1. The inner race 24 has a pair of oil passages 28a-b on opposite sides of the rollers. The sump housing 14 carries a plurality of windback seals 30a-c which co-operate with a corresponding plurality of seal runners 32a-c rotatable with the shaft 18. The seal runner 32c has an oil passage 34 therethrough.

The tubular end 16 of the rotor shaft 18 has an inside cylindrical wall 36 centred on the main axis 20. The inside wall 36 is bounded at the end of the shaft by a circumferential lip 38 which has an annular shoulder 40 facing the inside wall. At the other end, the inside wall 36 is bounded by an annular shoulder 42 between the wall 36 and a cavity 44 in a solid web 46 of the shaft.

The oil distribution system of the engine includes a pipe 48 carrying oil under pressure when the engine is operating. A pipe 50 branches from the pipe 48 and terminates at a nozzle 52 formed integrally with an end plate 54 attached to the sump housing 14. The pipe 50 conducts oil under

pressure to the nozzle 52 which has an orifice, not shown, adapted to direct a spray of oil toward the inside cylindrical wall 36 of the rotating shaft 18. The nozzle 52 limits the flow of oil toward the inside wall 36 to a rate equal to the combined rates required to lubricate and cool the bearing 22 and to cool the seal runners 32a-c. The oil spray is divided between the bearing and the seal runners by a metering system 56 according to this invention.

Referring to Figures 1-4, the metering system 56 includes a flow dividing insert 58 moulded from flexible and resilient fluorocarbon elastomer such as AMS 3216 manufactured by DuPont, U.S.A. and marketed under the trade name Viton. The insert 58 includes a cylindrical wall 60 having an outside diameter greater than the diameter of the inside cylindrical wall 36 of the shaft 18. The left-hand end, Figure 1, of the insert 58 is bounded by an annular end wall 62 the radial depth (i.e., thickness) of which is the same as the wall thickness of the cylindrical wall 60. The opposite or right-hand end of the insert is bounded by an annular end wall 64 which extends from the inside periphery of the cylindrical wall 60 to an intermediate cylindrical surface 66, Figure 4, having a smaller diameter than an outside cylindrical surface 68 of the insert. An annular shoulder 70 extends between the intermediate and outside cylindrical surfaces 66 and 68.

The cylindrical wall 60 of the insert has a diametrically-opposed pair of large perforations 72a-b and two diametrically-opposed pairs of small perforations 74a-b and 74c-d between the large perforations (see Figure 4). A first pair of diametrically-opposed lugs 76a-b, Figures 2 and 4, are formed on the intermediate cylindrical surface 66. A second pair of diametrically-opposed lugs 78a-b, Figures 1, 3 and 4, are formed on the outside surface 68 of the cylindrical wall 60 near the annular end 62 of the insert.

For installation, the insert 58 is aligned on the main axis 20 outside the hollow end of the shaft 18 and pressed through the circular opening defined by the lip 38. The flexibility and resilience of the insert permits it to be relatively easily pushed through the lip. The insert 58 is rotated relative to the shaft 18 until the lugs 76a-b register with respective ones of a pair of slots 80a-b, Figure 2, in the lip 38 of the shaft and is pushed to an installed position on the shaft. In the installed position, the end wall 62 seats against the shoulder 42 on the shaft and the annular end wall 64 springs out behind the annular shoulder 40. Concurrently, the lugs 78a-b on the insert seat in corresponding ones of a pair of radial bores 82a-b in the shaft, Figures 1 and 3.

In the installed position of the insert 58, Figure 1, the large and small perforations 72a-b and 74a-d, respectively, co-operate with the inside cylin-

dricall wall 36 of the shaft in defining a plurality of oil-catching pockets for capturing oil sprayed from the nozzle 52. For exhausting or draining the pockets, the metering system 56 includes a plurality of drain holes in the shaft within the confines of respective ones of the large perforations and the small perforations, only a single drain hole 84 being illustrated in Figure 1. The several drain holes are connected to individual ones of the oil passages 28a-b and 34 in the bearing inner race 24 and in the seal runner 32c, as for example by a plurality of schematically represented passages 86, Figure 1.

When the engine is operating, oil is sprayed from the nozzle 52 at a rate corresponding to the total flow required to cool the seal runners and to cool and lubricate the bearing. The flow capacity of the drain holes in the shaft exceeds the flow from the nozzle 52 so that the drain holes do not meter oil. Rather, the flow rate of oil through the individual drain holes is proportional to the areas of the large and small perforations 72a-b and 74a-d exposed to the spray. Thus, the oil flow to the rotating elements on the shaft connected to the pockets defined by the large perforations exceeds the oil flow to the rotating elements connected to the pockets defined by the small perforations. In addition, lugs 76a-b and 78a-b co-operate with slots 80a-b and the radial bores 82a-b, respectively, to prevent torsional distortion of the insert.

The flexibility and resilience of the insert 58 are important features of this invention. The flexibility permits relatively easy installation. With the outside diameter of the insert 58 greater than the diameter of the inside wall 36 on the shaft 18, the resilience of the insert assures self-retention of the insert on the shaft. Also, the resilience of the insert assures that the insert will self-seal against the inside wall 36 around the large and small perforations to prevent leakage between the oil-catching pockets.

Claims

1. A metering system (56) in a lubrication system for a gas turbine engine (12) having a rotatable shaft (18) with a hollow end portion (16) and an oil nozzle (52) operative to spray oil into said hollow end portion (16), said metering system (56) comprising: means on said shaft (18) defining an inside cylindrical wall (36) in said hollow end (16) of said rotatable shaft (18), and drainage means (86) provided in said inside cylindrical wall (36) to convey oil sprayed on to said inside cylindrical wall (36) to predefined portions of said gas turbine engine (12), characterised in that said metering system (56) includes a cylindrical insert (58) moulded of flexible and resilient fluorocarbon elastomer including a cy-

lindrical wall (60) having a first perforation (72a-b) therein and a second perforation (74a-b) therein and an outside diameter exceeding the diameter of said inside cylindrical wall (36) on said shaft (18), said insert (58) being disposed in said hollow end (16) of said shaft (18) and being self-retaining on said inside cylindrical wall (36) and being self-sealing against said inside cylindrical wall (36) around a first oil-catching pocket exposed to said oil spray which is defined by said first perforation (72a-b), and a second oil-catching pocket exposed to said oil spray which is defined by said second perforation (74a-b), and said drainage means on said shaft (18) includes a first drain (86) from said first oil-catching pocket and a second drain (86) from said second oil-catching pocket operative to exhaust oil from said first and said second oil-catching pockets at rates exceeding the rates at which said first and said second oil-pockets capture oil emitted from said oil spray (52).

2. A metering system according to claim 1, characterised in that there are means (76a-b,78a-b) provided on said insert (58) and on said rotatable shaft (18) which are operative to prevent torsional distortion of said insert (58) during rotation of said shaft (18).

3. A metering system according to claim 2, characterised in that said means on said insert (58) and on said rotatable shaft operative to prevent torsional distortion of said insert (58) include a plurality of lugs (76a-b,78a-b) moulded integrally with said insert (58) at opposite ends thereof, and a corresponding plurality of receptacles (80a-b,82a-b) on said rotatable shaft (18) receiving respective ones of said lugs (76a-b,78a-b) in said self-retained position of said insert (58) on said shaft (18).

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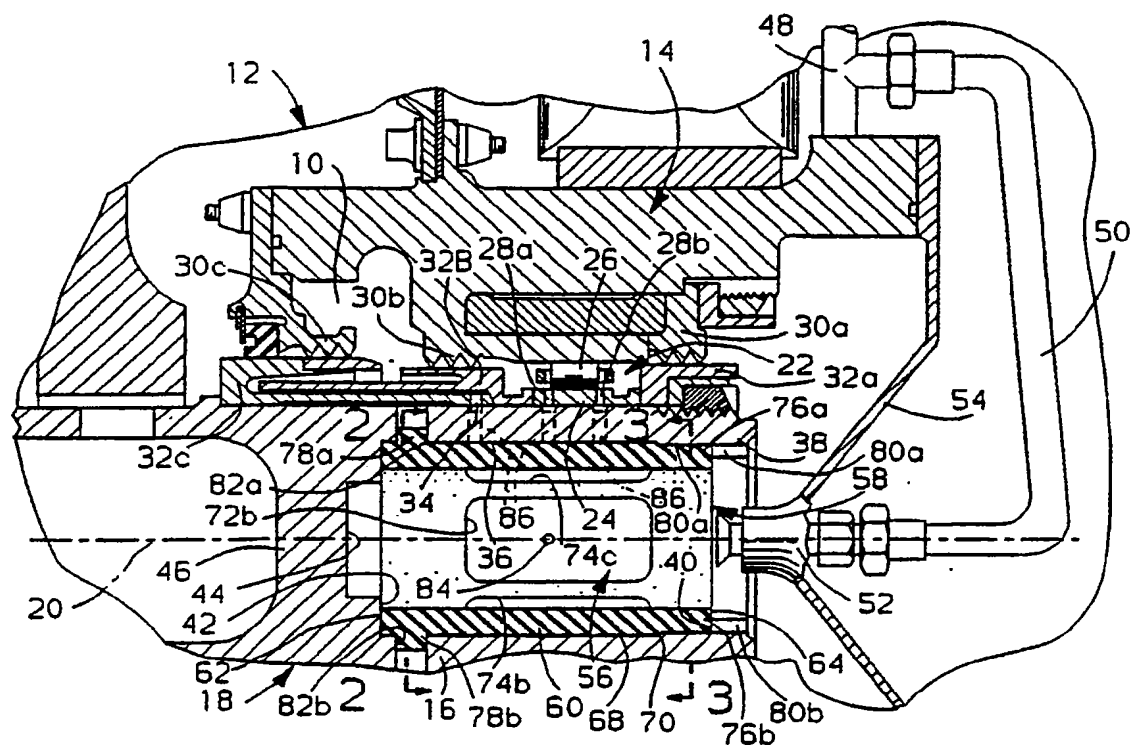


FIG. 1

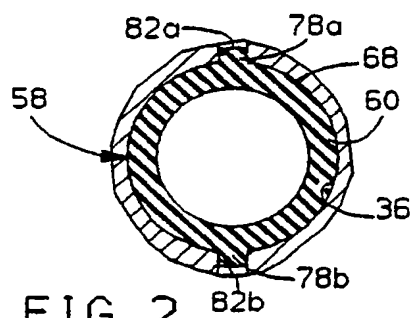


FIG. 2

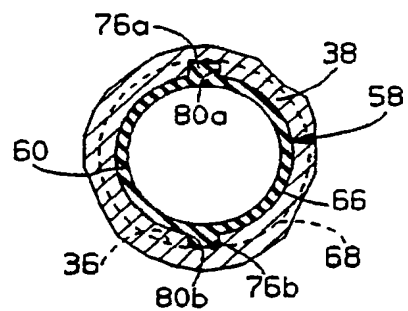


FIG. 3

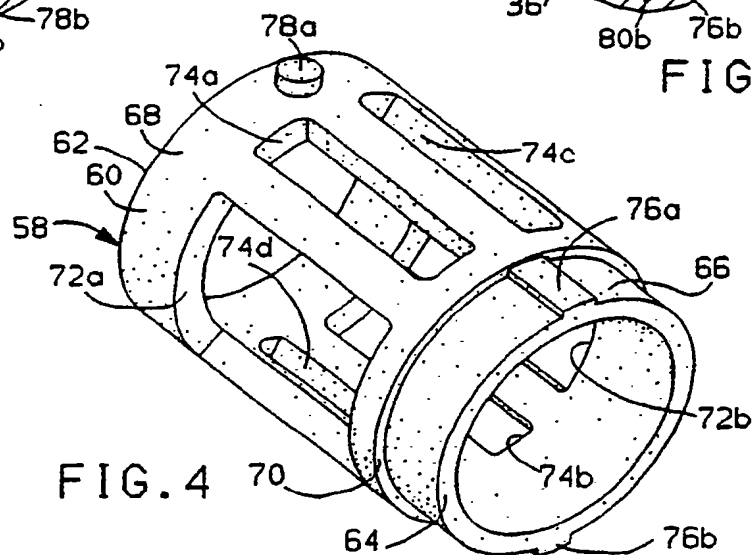


FIG. 4



European Patent
Office

EUROPEAN SEARCH REPORT

Application Number

EP 89 31 1818

DOCUMENTS CONSIDERED TO BE RELEVANT			
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int. Cl.5)
A	EP-A-90729 (SOCIETE NATIONALE D'ETUDE ET DE CONSTRUCTION DE MOTEURS D'AVIATION) * page 2, lines 15 - 30 * * page 6, line 8 - page 7, line 17; figure 1 *	1	F01D25/18 F02C7/06 F16C33/66
A	US-A-3960243 (D.J. DI ANTONIO)		
A	PATENT ABSTRACTS OF JAPAN vol. 10, no. 151 (M-483)(2208) 31 May 1986, & JP-A-61 6429 (YUBEA RUURON KOGYO K.K.) 13 January 1986, * the whole document *		
A	US-A-2999000 (MELVIN SPAT)		
			TECHNICAL FIELDS SEARCHED (Int. Cl.5)
			F01D F02C F16C F16N B65G
The present search report has been drawn up for all claims			
Place of search THE HAGUE		Date of completion of the search 06 JULY 1990	Examiner CRIADO Y JIMENEZ, F.
CATEGORY OF CITED DOCUMENTS			
X : particularly relevant if taken alone Y : particularly relevant if combined with another document of the same category A : technological background O : non-written disclosure P : intermediate document		T : theory or principle underlying the invention E : earlier patent document, but published on, or after the filing date D : document cited in the application I : document cited for other reasons & : member of the same patent family, corresponding document	

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